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A METHOD FOR DESIGNING DEEP UNDERGROUND STRUCTURES SUBJECTED TO DYNAMIC LOADS

by

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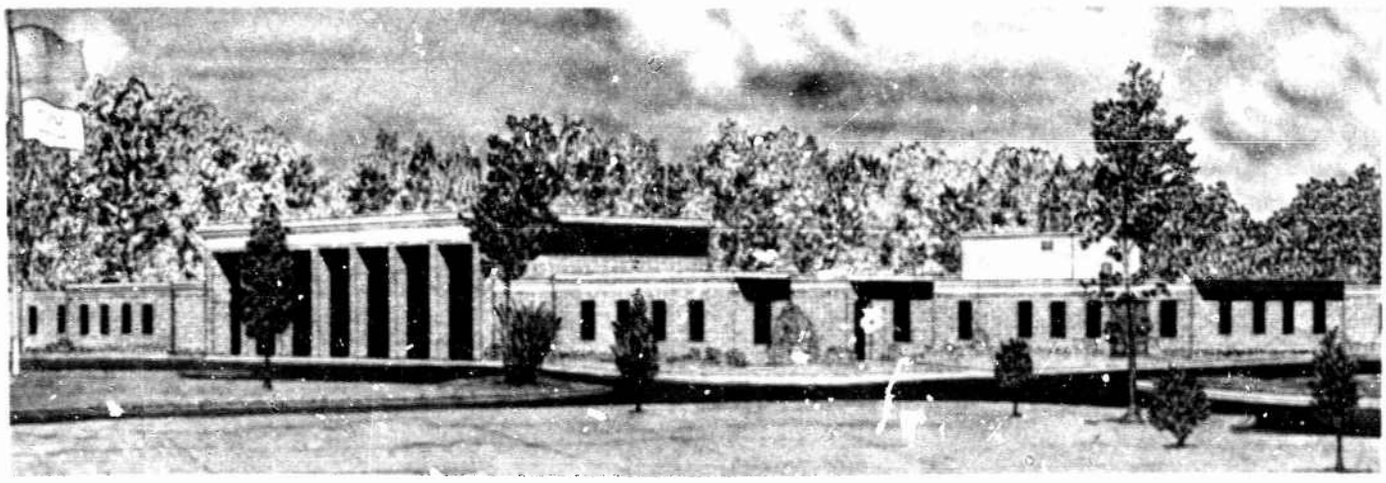

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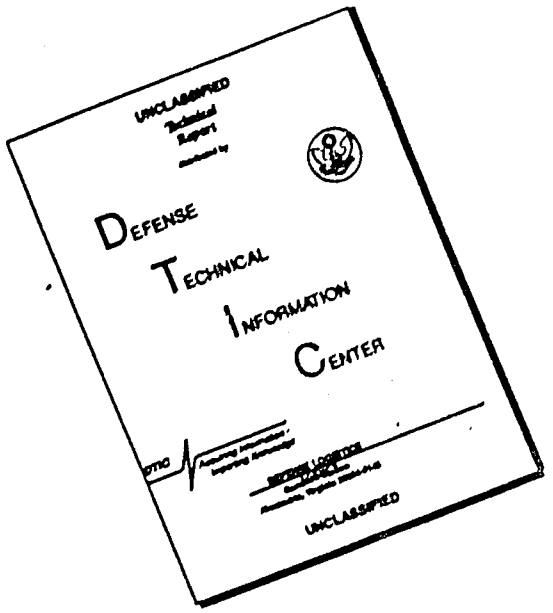

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This report describes solutions to a class of dynamic elastoplastic problems that model some of the salient features of the response of hardened underground facilities in rock. The theoretical model consisted of multilayered concentric cylinders of elastoplastic materials with time-dependent loads applied to the exterior boundary. Each element in the cross section was assumed to be incompressible and its yield governed by a Mohr-Coulomb failure criterion. The number of elements within the cross section was not limited. Solutions

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20. ABSTRACT (Continued).

of the theoretical model were cast in the general form normally used in structural dynamics:

Mass × Acceleration = External applied load - Internal resistance .

The resulting equations can be quickly and inexpensively evaluated on a digital computer.

To extend the range of validity of the exact theory, a first-order correction factor was developed to account for the compressibility of the materials, and a simple method to treat backpacked structures was introduced.

The theory was verified by comparing calculated values with experimental measurements from small-scale static and explosively driven tunnel collapse studies. Good agreement was noted for all cases considered.

PREFACE

This report presents results of a theoretical study which evaluates the response of hardened underground facilities in rock to dynamically applied loads produced by explosions. This research was conducted by personnel of the Phenomenology and Effects Division (PED) of the Weapons Effects Laboratory (WEL), U. S. Army Engineer Waterways Experiment Station (WES), during the period January 1975-January 1976.

The primary analytical development and preparation of the initial draft report were sponsored by the Defense Nuclear Agency under Subtask J34CAXSX311, "Underground Structures Studies," under the guidance of Dr. Kent Goering. Final report preparation and additional analytical work including investigations of backpacking and unloading and subsequent reloading were sponsored by the Office, Chief of Engineers, under Project 4A762719AT40/A1/017, "Stability of Deep Underground Structures in Rock," which was monitored by Mr. D. S. Reynolds.

This report was written by Messrs. J. L. Drake and J. R. Britt, PED, under the general supervision of Mr. L. F. Ingram, Chief, PED, and Mr. W. J. Flathau, Chief, WEL.

COL G. H. Hilt, CE, and COL J. L. Cannon, CE, were Directors of WES during the preparation and publication of this report. Mr. F. R. Brown was Technical Director.

CONTENTS

CONVERSION FACTORS, U. S. CUSTOMARY TO METRIC (SI)
UNITS OF MEASUREMENT

U. S. customary units of measurement used in this report can be converted to metric (SI) units as follows:

Multiply	By	To Obtain
mils	0.00254	centimetres
inches	2.54	centimetres
pounds (force) per square inch	6.894757	kilopascals
kilobars	100.00	megapascals
inches per second per second	2.54	centimetres per second per second
degrees (angle)	0.01745329	radians

A METHOD FOR DESIGNING DEEP UNDERGROUND STRUCTURES SUBJECTED TO DYNAMIC LOADS

CHAPTER 1

INTRODUCTION

1.1 BACKGROUND

Design of superhard underground facilities to resist the shock levels induced by nuclear weapons must take full advantage of the strength of the surrounding rock. Survival at levels in excess of 2 kbars[1] has been demonstrated by experiments in an idealized hard rock medium. In principle, additional hardness can be achieved by placing the facility deep enough to attentuate the shock to levels that can be resisted by structural hardening procedures.

Static methods are now being used for design of hardened underground facilities. Perhaps the best analytical procedure is that outlined by N. M. Newmark (Reference 1) wherein rock and liner systems are treated as elastoplastic materials that obey a general form of the Mohr-Coulomb failure criterion. Incompressible plastic strains and axially symmetric stresses are assumed. This analysis proceeds from the interior of the lining-medium system where the circumferential strain and radial stress are known, and successively works the solution outward by matching stresses and displacements across liner junctions and finally the rock-liner interface to obtain the free-field conditions in the rock. Similarly, it is possible to work from known conditions on any interior element inward or outward, but not from the free-field stress situation.

A. J. Hendron, Jr., and A. K. Aiyer (Reference 2) have extended Newmark's static analysis to include dilatancy of the Mohr-Coulomb material at failure. These authors report, however, that the increase in volume predicted by their theory is too large compared with experimental

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[1] A table of factors for converting U. S. customary units of measurement to metric (SI) units is presented on page 3.

evidence in real rocks. They claim that values of radial displacement calculated by their theory are a conservative upper bound. On the other hand, Newmark's analysis, which neglects any volume change due to the plastic strains at failure, predicts radial displacements that are too small and should be used as a lower bound.

Sophisticated computer codes are available that can account for the complex interactions of the structural elements under dynamic loading. General stress-strain relationships that closely model the real material behavior in both the elastic and inelastic regions can be used in these types of codes. However, these procedures are still too complex and time-consuming to be useful for design but should be useful for final analysis after a preliminary design has been proposed.

Treating the dynamic load as an equivalent static load is acceptable for use in structural design methods. In the case of normal structural elements (columns, beams, etc.), the equivalent static loading may be determined from a dynamic response chart for various ratios of the duration of loading to the natural period of the structure. For rock tunnel and liner systems the natural period is not easily defined without a dynamic analysis of the entire system.

1.2 OBJECTIVE

The goal of this report is to develop solutions for a class of dynamic problems that have direct bearing on design considerations for deep underground facilities in rock. Specifically, an extension of the static methods outlined by Newmark to accommodate dynamic loads is sought.

1.3 APPROACH

This report describes solutions to a class of dynamic elastoplastic problems that model some of the salient features of the response of hardened underground facilities in rock. The theoretical model consists of multilayered concentric cylinders of elastoplastic materials with time-dependent loads applied to the exterior boundary. Each element in the cross section is assumed to be incompressible and its yield governed by a Mohr-Coulomb failure criterion. The number of elements within the

cross section is not limited. Solutions of the theoretical model are cast in the general form normally used in structural dynamics:

Mass × Acceleration = External applied load - Internal resistance

The resulting equations can be quickly and inexpensively evaluated on a digital computer.

To extend the range of validity of this exact theory, a first-order correction factor was developed to account for the compressibility of the materials, and a simple method to treat backpacked structures was introduced. The theory was then verified by comparing calculated values with experimental measurements from small-scale static and explosively driven tunnel collapse studies.

CHAPTER 2

THEORY

2.1 PROBLEM FORMULATION

The rock and liner system is modeled as multilayered concentric cylinders of elastoplastic materials in plane strain (see Figure 2.1). An arbitrary time-dependent pressure is applied to the outer boundary, and an arbitrary pressure, also time-dependent if desired, is applied to the internal boundary. It is assumed that the materials in the cross section are incompressible in both the elastic and the plastic states. (As discussed in Section 2.3, a first-order compressibility correction is easily incorporated.) With these assumptions, the effects of material compaction and stress wave interaction within the cross section are neglected, thereby greatly simplifying the analysis. The number of material layers in this analysis is unlimited.

2.1.1 Field Equations. Equations requiring equilibrium and conservation of mass are basic to any analytical study of structural response. Solutions to these equations are valid in both the elastic and plastic states of the material.

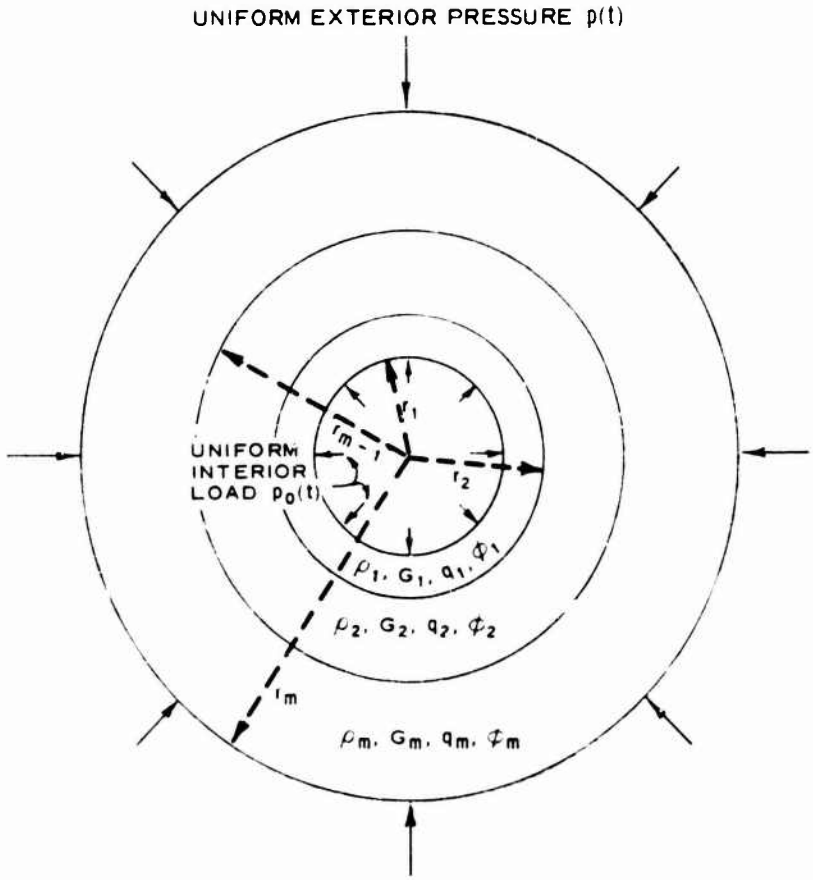
The equation of motion in polar coordinates is as follows:[1]

$$\frac{\partial \sigma_r^i}{\partial r} + \frac{\sigma_r^i - \sigma_\theta^i}{r} = \rho_i \left(\frac{\partial^2 u_r^i}{\partial t^2} \right) \qquad (i = 1,2,\ldots,m) \qquad (2.1)$$

where

σ_r^i = radial stress component
r = radial coordinate
σ_θ^i = tangential stress component
ρ_i = mass density
u_r^i = radial displacement component
t = time

[1] For convenience, symbols and unusual abbreviations are listed and defined in the Notation (Appendix A).



LEGEND

r_1, r_2 = INTERIOR BOUNDARIES OF THE FIRST AND SECOND MATERIAL LAYERS, RESPECTIVELY

m = NUMBER OF MATERIALS IN THE CROSS SECTION

ρ = DENSITY

G = SHEAR MODULUS

q = COHESION

Φ = ANGLE OF INTERNAL FRICTION

Figure 2.1 Geometry of rock-liner system.

i = subscript or superscript referring to a component material in the cross section

m = number of materials in the cross section

This equation can be solved when relationships between stresses and displacements (constitutive laws) are given. In the plastic state an additional equation given by the yield criterion relating the radial and tangential stresses is necessary for a complete solution.

Incompressibility requires the conservation of mass to be

$$\frac{\partial u_r^i}{\partial r} + \frac{u_r^i}{r} = 0 \qquad (i = 1,2,\ldots,m) \tag{2.2}$$

The general solution to Equation 2.2 is

$$u_r^i = \frac{U^i(t)}{r} \tag{2.3}$$

where $U^i(t)$ is a general function of time to be determined from the boundary conditions. It is evident from Equation 2.3 that if the displacement field is continuous between component materials, $U^i(t)$ must be the same for all materials. Thus, the superscript i is dropped such that

$$U^1(t) = U^2(t) = \ldots = U(t) \tag{2.4}$$

2.1.2 Material Models. Stress-strain relationships for an incompressible elastic medium in plane strain can be written as

$$\begin{aligned} \sigma_r^i &= S^i + 2G_i\epsilon_r^i \\ \sigma_\theta^i &= S^i + 2G_i\epsilon_\theta^i \\ \sigma_z^i &= S^i \end{aligned} \tag{2.5}$$

where

S^i = mean normal stress

G_i = shear modulus

ε_r^i = radial strain component

ε_θ^i = tangential strain component

σ_z^i = axial stress component

and the mean normal stress is given by

$$S^i = \frac{1}{3}\left(\sigma_r^i + \sigma_\theta^i + \sigma_z^i\right)$$

In polar coordinates, the strains are defined as

$$\varepsilon_r^i = \frac{\partial u_r^i}{\partial r} = -\frac{U(t)}{r^2}$$
$$\varepsilon_\theta^i = \frac{u_r^i}{r} = \frac{U(t)}{r^2} \qquad (2.6)$$

where the radial displacement given by Equation 2.3 is substituted into the strain definitions. Because $U(t)$ is independent of the state of the material, it is evident that Equations 2.6 are valid in both the elastic and plastic regions.

Stress-strain relations in the plastic region are implied and defined by the condition of incompressibility and from the form of the yield law. A Mohr-Coulomb yield criterion assumed for all the component materials is given as

$$\left(\sigma_r^i - \sigma_\theta^i\right) = \sigma_r^i(1 - N_i) + 2q_i\sqrt{N_i} \qquad (2.7)$$

where

$$N_i = \tan^2\left(45° + \frac{\phi_i}{2}\right) = \frac{1 + \sin\phi_i}{1 - \sin\phi_i}$$

and q_i is the cohesion and ϕ_i is the angle of internal friction. The assumption of incompressibility is not strictly compatible with this yield criterion except for $\phi_i = 0$ (von Mises yield criteria) unless a nonassociative flow rule is assumed. Mohr-Coulomb materials dilate while undergoing failure; however, neglecting this effect is not

expected to affect the results greatly, and simplifies the solution.

2.1.3. Boundary Conditions. Stresses and displacements are assumed to be continuous across the various internal boundaries between materials. Further, dynamic pressures are applied at the exterior and interior boundaries. The external pressure was assumed to be much greater than the interior pressure, causing the deformation to be inward. These boundary conditions can be expressed as

$$\begin{aligned}
&\sigma_r^1 = -p_o(t) \quad \text{at} \quad r = r_1 \\
&\left.\begin{aligned}\sigma_r^i &= \sigma_r^{i+1} \\ u_r^i &= u_r^{i+1}\end{aligned}\right\} \quad \text{at} \quad r = r_{i+1} \qquad (i = 1,2,\ldots,m-1) \\
&\sigma_r^m = -p(t) \quad \text{at} \quad r = r_{m+1}
\end{aligned} \tag{2.8}$$

where

$p_o(t)$ = internal pressure (compressive)

r_i = interior boundary of the i^{th} material layer

$p(t)$ = external pressure-time history (compressive)

all of which are depicted in Figure 2.1.

Additional internal boundary conditions may occur if any of the component materials are partially elastic and partially plastic. Across this elastic-plastic boundary

$$\left(\sigma_r - \sigma_\theta\right)^i_{\text{elastic}} = \left(\sigma_r - \sigma_\theta\right)^i_{\text{plastic}}$$

and the normal stresses σ_r^i are continuous.

2.2 PROBLEM SOLUTIONS

Solutions to the equation of motion (Equation 2.1) are possible from the displacement solution (Equation 2.3) and either the stress-strain relations in the elastic region (Equation 2.5) or the yield criterion (Equation 2.7) in the plastic region.

2.2.1 Elastic Region. In the elastic region, the following

equation is obtained from the elastic stress-strain relations and from Equations 2.6:

$$\sigma_r^i - \sigma_\theta^i = -4G_i\left[\frac{U(t)}{r^2}\right]$$

Substituting this expression into the equation of motion (Equation 2.1) gives

$$\frac{\partial\sigma_r^i}{\partial r} - 4G_i\left[\frac{U(t)}{r^3}\right] = \frac{\rho_i}{r}\left[\frac{d^2U(t)}{dt^2}\right]$$

with the solution

$$\sigma_r^i = \rho_i \ln\left(\frac{r}{r_i}\right)\left[\frac{d^2U(t)}{dt^2}\right] - \frac{2G_i}{r_i^2}\left(\frac{r_i}{r}\right)^2 U(t) + C_i \qquad (2.9)$$

where C_i is a constant independent of r to be determined from boundary conditions. From this result and Equations 2.5 and 2.6 the mean normal stress S^i is given by

$$S^i = \sigma_r^i + \left(\frac{2G_i}{r_i^2}\right)\left(\frac{r_i}{r}\right)^2 U(t) \qquad (2.10)$$

2.2.2 Plastic Region. An expression for the stress difference $\left(\sigma_r^i - \sigma_\theta^i\right)$ in the plastic region is given by the yield condition (Equation 2.7). Substituting this relationship into the equation of motion (Equation 2.1) yields

$$\frac{\partial\sigma_r^i}{\partial r} + \frac{\sigma_r^i}{r}(1 - N_i) = \frac{1}{r}\left\{\rho_i\left[\frac{d^2U(t)}{dt^2}\right] - 2q_i\sqrt{N_i}\right\}$$

with the solution

$$\sigma_r^i = \left\{\rho_i\left[\frac{d^2U(t)}{dt^2}\right] - 2q_i\sqrt{N_i}\right\}\left(\frac{1}{1 - N_i}\right)\left[1 - \left(\frac{r}{r_i}\right)^{N_i-1}\right] + C_i\left(\frac{r}{r_i}\right)^{N_i-1} \qquad (2.11)$$

Note that

$$\lim_{n_i \to o} \quad \frac{1}{n_i}\left[\left(\frac{r}{r_i}\right)^{n_i} - 1\right] = \ln\left(\frac{r}{r_i}\right)$$

$$\lim_{n_i \to o} \quad \left(\frac{r}{r_i}\right)^{n_i} = 1$$

where $n_i = N_i - 1$. Thus, in the limit as $N_i \to 1$ or $\phi_i \to 0$, the expressions for the elastic stress distribution (Equation 2.9) and plastic distributions (Equation 2.11) bear similar terms for the inertial contribution and for the arbitrary constant C_i .

2.2.3 Generalized Stress Distribution. Since each material is allowed to be elastic or plastic or even partially yielded, the number of stress combinations required to solve the boundary conditions for the constant C_i is m^2 for m materials in the cross section. This situation can be avoided by writing a more general stress distribution which can be used in both the elastic and plastic regions. This generalized stress distribution is expressed in operator form as

$$\sigma_r^i = \left\{\frac{\rho_i}{n_i}\left[\left(\frac{r}{r_i}\right)^{n_i} - 1\right]\frac{d^2}{dt^2} - \frac{2G_i}{r_i^2}\left(\frac{r_i}{r}\right)^2\right\}U(t)$$

$$- \left(\frac{2q_i}{n_i}\right)\sqrt{(n_i + 1)}\left[\left(\frac{r}{r_i}\right)^{n_i} - 1\right] + C_i\left(\frac{r}{r_i}\right)^{n_i} \qquad (2.12)$$

where

$$n_i = N_i - 1 = \frac{2 \sin \phi_i}{1 - \sin \phi_i}$$

and again noting

$$\lim_{n_i \to o} \quad \frac{1}{n_i}\left[\left(\frac{r}{r_i}\right)^{n_i} - 1\right] = \ln\left(\frac{r}{r_i}\right)$$

In the elastic state it is necessary to define $n_i = q_i = 0$ for Equation 2.12 to be valid. The plastic state requires $G_i = 0$ to obtain the

correct stresses. Using Equation 2.12 in the boundary conditions will yield a generalized form for the constant C_i which is valid in either the elastic or plastic state and which correctly accounts for the states of all m elements.

2.2.4 Stress Solution. Substituting Equation 2.12 into the boundary conditions expressed by Equations 2.8 gives the following recurrence formulas for C_i :

$$C_1 = -p_o(t) + \frac{2G_1}{r_1^2} U$$

$$C_{i+1} = \frac{\rho_i}{n_i}\left[\left(\frac{r_{i+1}}{r_i}\right)^{n_i} - 1\right] \frac{d^2U}{dt^2} - \frac{2G_i}{r_i^2}\left(\frac{r_i}{r_{i+1}}\right)^2 U + \frac{2G_{i+1}U}{r_{i+1}^2}$$

$$- \frac{2q_i}{n_i}\sqrt{n_i + 1}\left[\left(\frac{r_{i+1}}{r_i}\right)^{n_i} - 1\right] + C_i\left(\frac{r_{i+1}}{r_i}\right)^{n_i} \qquad (2.13)$$

Completing this solution and continuing the recurrence out to $r = r_{m+1}$ gives the following formula:

$$\sigma_r^i = \left\{\frac{\rho_i}{n_i}\left[\left(\frac{r}{r_i}\right)^{n_i} - 1\right] + \left(\frac{r}{r_i}\right)^{n_i} M_{i-1}\right\} \frac{d^2U}{dt^2}$$

$$+ \left\{\frac{2G_i}{r^2}\left[\left(\frac{r}{r_i}\right)^{n_i+2} - 1\right] + \left(\frac{r}{r_i}\right)^{n_i} K_{i-1}\right\} U$$

$$- \left\{\left(\frac{2q_i}{n_i}\right)\sqrt{n_i + 1}\left[\left(\frac{r}{r_i}\right)^{n_i} - 1\right] + \left(\frac{r}{r_i}\right)^{n_i}\left(Y_{i-1} + P_{oi-1}\right)\right\} \qquad (2.14)$$

where

$$M_i = \sum_{j=1}^{i} \frac{\rho_j}{n_j} \left[\left(\frac{r_{j+1}}{r_j} \right)^{n_i} - 1 \right] \prod_{k=j+1}^{i} \left(\frac{r_{k+1}}{r_k} \right)^{n_k}$$

$$Y_i = \sum_{j=1}^{i} \left\{ \left(\frac{2q_i}{n_j} \right) \sqrt{n_j + 1} \left[\left(\frac{r_{j+1}}{r_j} \right)^{n_j} - 1 \right] \right\} \prod_{k=j+1}^{i} \left(\frac{r_{k+1}}{r_k} \right)^{n_k}$$

$$K_i = \sum_{j=1}^{i} \frac{2G_j}{r_{j+1}^2} \left[\left(\frac{r_{j+1}}{r_j} \right)^{n_j+2} - 1 \right] \prod_{k=j+1}^{i} \left(\frac{r_{k+1}}{r_k} \right)^{n_k}$$

$$P_{oi} = p_o(t) \prod_{k=1}^{i} \left(\frac{r_{k+1}}{r_k} \right)^{n_k} \tag{2.15}$$

and

$$M\left(\frac{d^2U}{dt^2} \right) = \left[P_o(t) - p_o(t) \right] - (KU - Y) \tag{2.16}$$

where

$M = M_m$ = effective mass

$P_o = P_{om}$ = effective interior pressure

$K = K_m$ = effective elastic stiffness

$Y = Y_m$ = effective rigid plastic collapse pressure of the m materials

The symbol $\sum_{j=1}^{i}$ denotes a sum of terms with $j = 1$ to $j = i$, and

$\prod_{k=j+1}^{i}$ denotes a product of terms with $k = j + 1$ to $k = i$.

Equations 2.14-2.16 along with the yield condition (Equation 2.7) completely describe the stresses and motions of the rock-liner system

under dynamic loading. The differential Equation 2.16 is of the same form as that used in structural dynamics:

$$\text{Mass} \times \text{Acceleration} = \text{External applied load} - \text{Internal resistance}$$

In this case, the internal load-resistance function is $R = KU - Y$. Note that initially $Y = 0$ and the resistance is due to the elastic stiffness K. As the external applied load increases, elements begin to fail, effectively increasing Y as K decreases until the applied load is such that the entire section fails, the full value of Y is reached, and $K = 0$.

2.2.5 Elastic-Plastic Boundaries. Solutions expressed by Equations 2.14 and 2.16 are valid for all load conditions in both the elastic and plastic ranges as long as the deformations are inward; however, these equations alone are not sufficient to describe the stresses and motions of the rock-liner system. Yield conditions must be used along with these equations to determine the elastic-plastic state of the system.

When the deformation is inward, yielding in each of the component materials begins at the interior boundary and progresses outward until the entire element is fully plastic. This behavior can be used to simplify the analysis as follows. Divide each component material into two parts, the inner portion with only plastic properties and the outer shell with elastic properties. With this definition the m materials are divided into two m layers where

$$\left.\begin{matrix} n_i \\ q_i \end{matrix}\right\} = 0 \quad \text{for even values of } i$$

$$G_i = 0 \quad \text{for odd values of } i \qquad (2.17)$$

and r_{i+1} for odd values of i is the elastic-plastic interface. A material is fully elastic when $r_{i+1} = r_i$ for odd values of i and

a material is fully plastic when $r_{i+1} = r_{i+2}$ for odd values of i. Using these definitions, the elastic-plastic boundary, r_{i+1} for odd values of i, can be calculated from the yield conditions.

Continuity of stresses across the elastic-plastic interface requires that

$$\left(\sigma_r^{i+1} - \sigma_\theta^{i+1}\right) = \left(\sigma_r^i - \sigma_\theta^i\right) \qquad \text{at} \quad r = r_{i+1} \quad \text{for odd values of} \quad i$$

Using Equations 2.5 and 2.6 in the elastic range and Equation 2.7 in the plastic region gives

$$-\frac{4G_{i+1}}{r_i^2}\left(\frac{r_i}{r_{i+1}}\right)^2 U = -n_i\sigma_r^i + 2q_i\sqrt{n_i + 1} \qquad \text{at} \quad r = r_{i+1}$$

$$\text{for odd values of} \quad i$$

Substituting in this result the solution for radial stresses (Equation 2.14) yields the expression for r_{i+1}/r_i and i = odd

$$-\frac{4G_{i+1}}{r_i^2}\left(\frac{r_i}{r_{i+1}}\right)^2 U = -\left(\frac{r_{i+1}}{r_i}\right)^{n_i}\left[\left(\rho_i + n_i M_{i-1}\right)\left(\frac{d^2U}{dt^2}\right)\right.$$

$$\left. + n_i\left(K_{i-1}U - Y_{i-1} - P_{oi-1}\right) - 2q_i\sqrt{n_i + 1}\right] + \rho_i\left(\frac{d^2U}{dt^2}\right) \qquad (2.18)$$

For $n_i = 0$ this equation reduces to the simple form

$$\left(\frac{r_{i+1}}{r_i}\right)^2 = \frac{-2G_{i+1}U}{q_i r_i^2} \qquad (2.19)$$

Note that for the Mohr-Coulomb failure criterion, inertial mass × acceleration terms appear in the condition for dynamic failure. Until the time at which the maximum inward particle velocity is reached, when $d^2U/dt^2 \leq 0$, the inertial forces tend to strengthen the section causing r_{i+1}/r_i to be less than its static value. However, when $d^2U/dt^2 > 0$,

these forces cause the plastic boundary to grow outward more rapidly than under static load.

2.3 A FIRST-ORDER COMPRESSIBILITY CORRECTION

For static loading and elastic response the ratio of the compressible to the incompressible radial displacements is (Reference 3)

$$\frac{u_c}{u_i} = 1 + (1 - 2\nu)r^2\left[\frac{r_2^2p_2 - r_1^2p_1}{r_1^2r_2^2(p_2 - p_1)}\right]$$

where ν is Poisson's ratio of the material, r_1 and p_1 are the inside radius and applied load, respectively, and r_2 and p_2 are the outside radius and applied load, respectively. For most tunnel liners being considered, $p_1 = 0$ (atmospheric pressure) so that p_1/p_2 is very small. Expanding the expression for u_c/u_i evaluated at $r = r_1$ in powers of p_1/p_2 gives

$$\frac{u_c(r_1)}{u_i(r_1)} = 1 + (1 - 2\nu)\left\{1 + \left[1 - \left(\frac{r_1}{r_2}\right)^2\right]\sum_{n=1}^{\infty}\left(\frac{p_1}{p_2}\right)^n\right\}$$

For $p_1/p_2 = 0$ this expression becomes

$$\frac{u_c(r_1)}{u_i(r_1)} = 2(1 - \nu) \qquad (2.20)$$

When p_1 is zero, this ratio is exact for the static elastic case. Experience indicates that this is also a reasonably accurate, first-order correction factor for the dynamic displacements of an elastic-plastic rock-tunnel liner system. In this application an average value of the Poisson's ratios of the component materials should be used. Typically, ν for common rock and liner materials ranges from about 0.2 to 0.3, and $u_c(r_1)/u_i(r_1)$ ranges from 1.6 to 1.4. Average values of $\nu = 0.25$ and $u_c(r_1)/u_i(r_1) = 1.5$ are adequate for most calculations.

2.4 A SUMMARY OF THE CALCULATIONAL METHOD

Equations 2.18 or 2.19 together with the equation of motion, 2.16, can be solved for $U(t)$ by a numerical integration such as the Runge-Kutta method. Then using Equation 2.6 and the compressibility correction, Equation 2.20, the radial displacement can be determined. The radial stress component is given by Equations 2.14 and 2.15; then with the mean normal stress S^i given in Equation 2.10, the other stress components are given in Equations 2.5 and 2.7. Thus, these relations provide a complete description of the stresses and motions of the rock-liner system in a form suitable for fast computer calculations.

2.5 TREATMENT OF BACKPACKED STRUCTURES

Backpacking is commonly a crushable material, such as foamed concrete, placed between the rock and the tunnel liner to absorb the large strains suffered by the rock and, consequently, to isolate the liner from excessive damage. The simplest treatment of backpacking is the hydrostatic model, which neglects the shear strength of the material. Thus the pressures at the inside and outside of the backpacking are the same. Experimental pressure-volumetric strain curves can be used in this model to calculate the pressure on the rock and liner. A simple iterative process can be set up which links the radial displacement of the liner to the displacement of the rock as a function of volume changes in the backpacking. Since the present analysis allows a time-varying interior pressure $p_o(t)$, the calculational method is easily adapted for computations using this hydrostatic model of backpacking. The computer code presently in use at the U. S. Army Engineer Waterways Experiment Station includes this option.

For free-field stresses in the rock which are essentially axisymmetric (uniform radial load), the hydrostatic model provides an accurate description of the backpacking collapse. The effect of shear strength of the backpacking on rock-liner displacements is negligibly small in this case.

For the biaxial free-field stresses in which the difference between the vertical and horizontal normal stress components is large compared

with the collapse strength of the backpacking, the hydrostatic model of collapse of the backpacking is no longer valid for relatively large rock deformations. Since the material cannot flow to equalize the stresses, ovaling of the backpacking results, transmitting highly nonuniform radial loads to the liner which may collapse prematurely.

2.6 DISCUSSION OF UNLOADING

The present analysis is limited to monotonically increasing displacements. In the case of small plastic strains, unload and a subsequent reload may take place elastically which can be treated simply within the present analysis. For von Mises materials in static loading, Prager and Hodge (Reference 3) give equations for both elastic and plastic unload. The plastic unload of a Mohr-Coulomb material may be treated in a similar manner; but the analysis is more complex than the von Mises case, as the authors of the present report have found in a thorough study of this process. In the present study it was discovered that the Mohr-Coulomb plastic unloading often involves further yielding of the material even as the inward displacement decreases. The manner and extent of this unload yielding depend primarily on (1) the amount of plastic deformation in the material at the end of the load phase and (2) the confining pressure at the interior of the layer of material. The resulting equations are mathematically too cumbersome to be included in the analysis of this report. The experience gained in this study indicates that elastic unload is often a good first approximation for the purposes of this report, but an accurate theoretical description of repeated inelastic loading and unloading of Mohr-Coulomb materials must be more rigorous and will require further theoretical work.

CHAPTER 3

EVALUATION OF THE THEORY

3.1 COMPARISON OF CALCULATIONS WITH EXPERIMENTAL EVIDENCE

Calculations were made for small-scale tests conducted by the Stanford Research Institute (SRI) (Reference 4) to evaluate the theory of this report. In these experiments explosives were used in a specially designed device to produce axially symmetric loads. Displacements as a function of time were measured using the "light ring" technique which consists of photographing the motion of the inner pipe wall by means of a bright ring of light transmitted through a transparent flexible plastic tube glued to the pipe wall. The targets considered for calculation were 3-inch-OD tubes of "super lean grout" (SLG) lined with a 10-mil-thick pipe of Type 347 stainless steel. The material properties used in the calculations are given in Table 3.1.

Figure 3.1 shows data from two shots with 80 grams of explosive. The estimated free-field pressure pulse obtained from calibration tests is also shown in the figure. Calculated curves are for Poisson's ratio, $\nu = 0.25$, 0.38 , and 0.5 , where 0.38 was the value of ν calculated from ultrasonic velocity measurements in SLG given in Reference 5. There is good agreement between the theory and experiment. Figure 3.2 presents measurements from three shots with estimated free-field pressure pulses as shown. Here the calculated values are close to the experimental points, but the agreement is not as good as in Figure 3.1.

An additional check of the theory can be obtained by noting that the internal resistance $R = KU - Y$ is the static stress-strain curve. Thus data from static or quasi-static tests can be used to evaluate this important part of the dynamic equation of motion (Equation 2.16). Figure 3.3 shows static calculations compared with measurements obtained by SRI.[1] In this test a hydrostatic load was applied to a sample of

[1] Letter of 10 Jun 1975 from T. C. Kennedy, Stanford Research Institute, Menlo Park, Calif., to J. Drake, Weapons Effects Laboratory, U. S. Army Engineer Waterways Experiment Station, CE, Vicksburg, Miss.

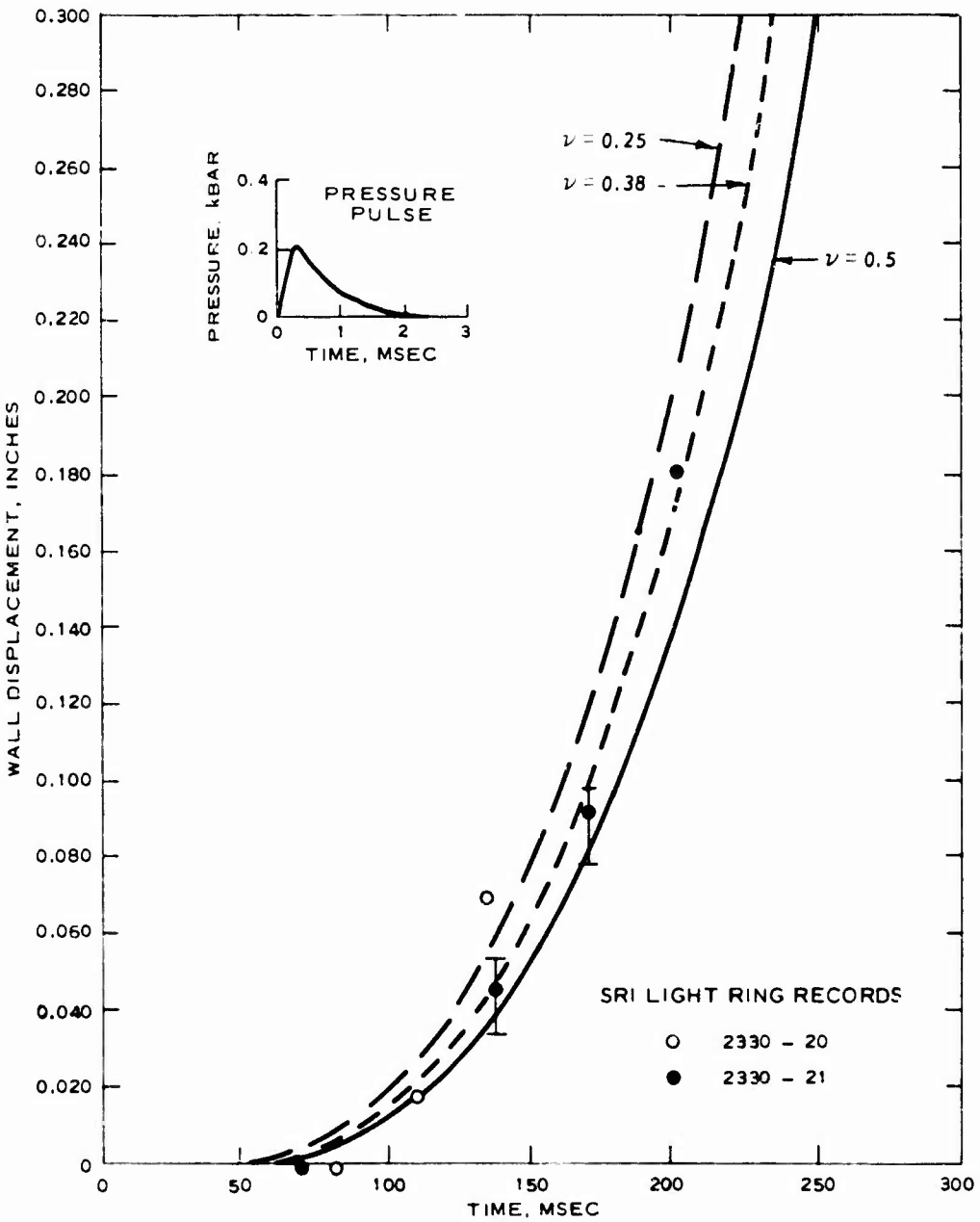


Figure 3.1 Calculated displacement-time curves compared with measurements with 80-gram charges (from Reference 4).

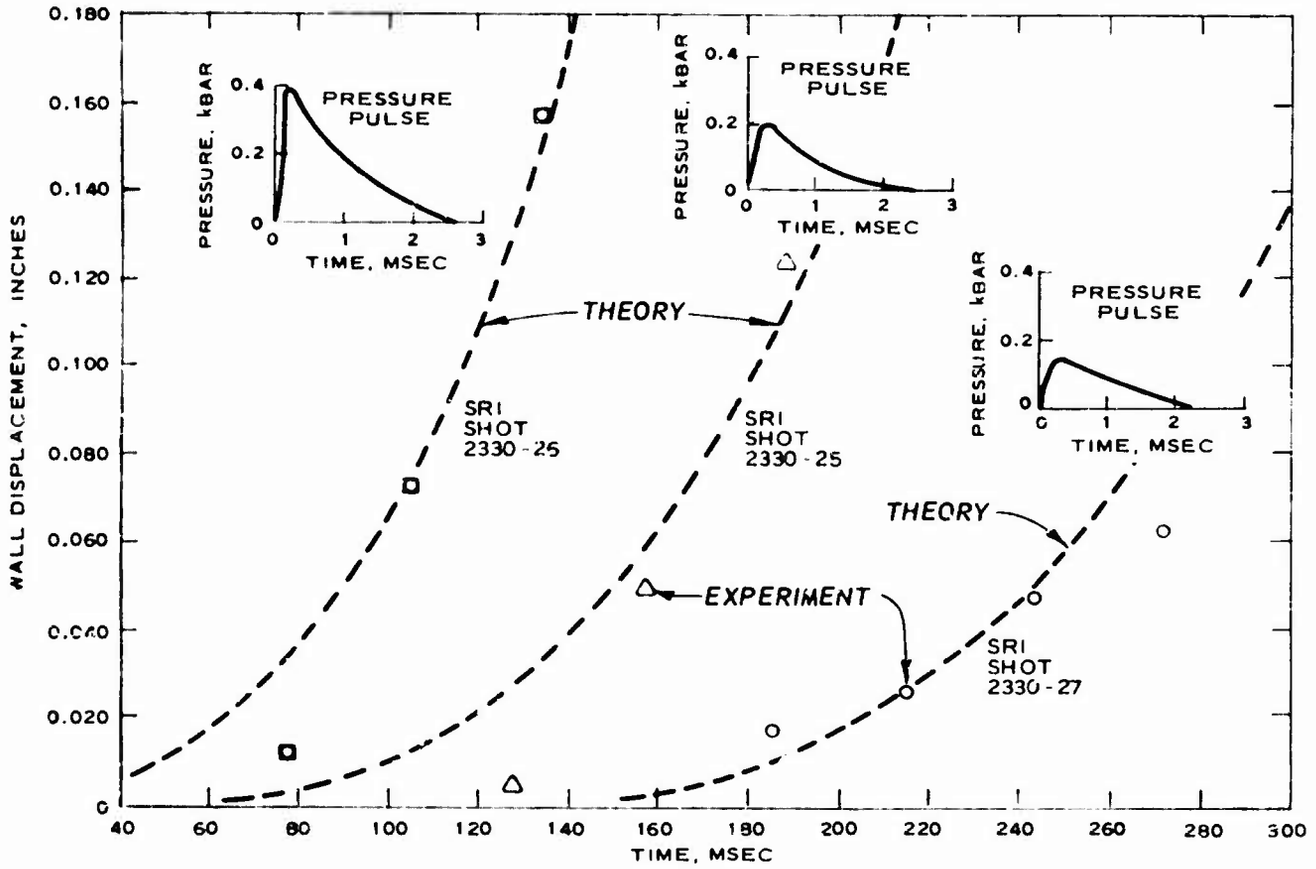


Figure 3.2 Theoretical and experimental wall displacements versus time for three pressure pulses (from Reference 4).

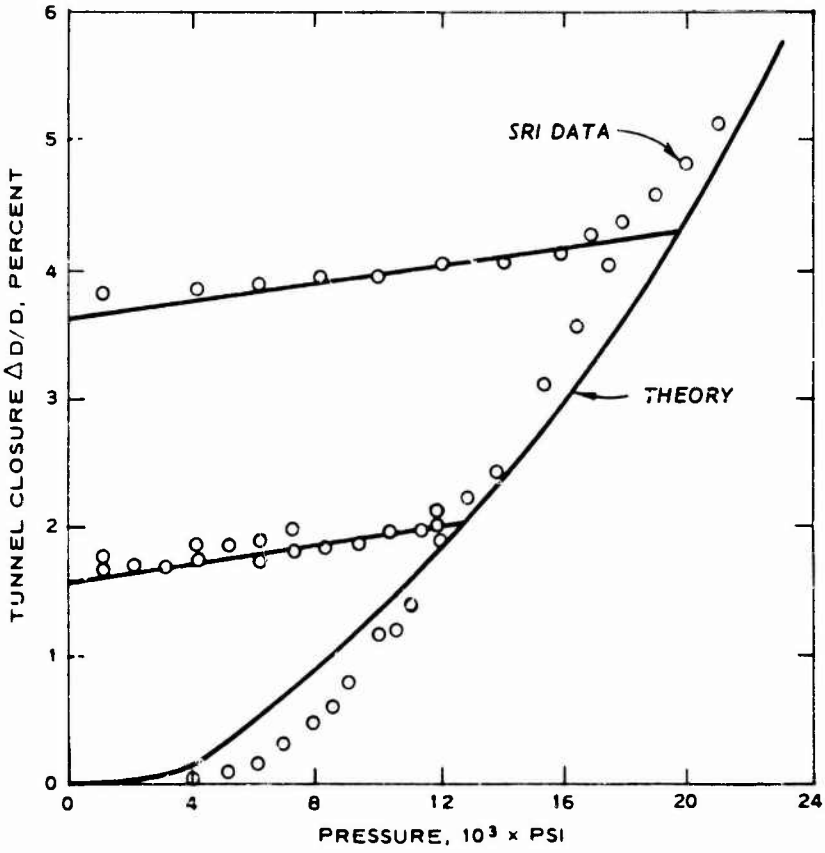


Figure 3.3 Computed and measured static tunnel closure versus pressure.

WES 6B rock simulant containing a 0.625-inch-ID tunnel lined with a 0.025-inch-thick mild steel tube. The material properties used in the calculation are given in Table 3.1. The displacement or tunnel closure is expressed as a percentage of the inside diameter D of the tunnel. The calculated unload curves assumed elastic behavior. Again, there is good agreement between theory and experiment.

3.2 CONCLUSIONS

An analysis has been presented which extends the static methods outlined by Newmark (Reference 1) to accommodate dynamic loads. Comparisons of calculations with experimental data have demonstrated that the analysis will predict satisfactorily the time-displacement curve in uniform radial load for materials which can be modeled as elastic-plastic

media having a Mohr-Coulomb or von Mises yield criterion. The analysis given in this report is in a form which allows quick, inexpensive preliminary design calculations for deep underground structures.

Users of the theory must be warned, however, that the analysis will not predict accurately displacements for biaxial loads where the vertical and horizontal normal stress components are significantly different or where a large amount of ovaling of the tunnel liner may be expected. In this case the theory will provide a conservative lower bound estimate of the deformation. At present, a design method for biaxial loading is not available in a relatively simple form comparable with the analysis procedure outlined in this report. Work in this area is needed.

The present analysis is limited to monotonically increasing displacements. In the case of small plastic strains, unload and a subsequent reload may take place elastically. This process can be treated simply within the present analysis.

TABLE 3.1 MATERIAL PROPERTIES USED IN THE CALCULATIONS

Material	Shear Modulus G 10^6 psi	Poisson's Ratio ν	Cohesion q 10^3 psi	Angle of Internal Friction ϕ degrees	Density ρ 10^{-4} psi $\frac{}{\text{in./sec}^2}$
Super lean grout	0.058	0.38	0.016	0.7	1.65
347 stainless steel	11.2	0.3	20	0	7.5
WES 6B rock simulant	0.48	0.25	1.7	30	--
Mild steel	12	0.3	20	0	--

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3. W. Prager and P. G. Hodge; "Theory of Perfectly Plastic Solids"; 1951; John Wiley and Sons, New York, N. Y.

4. R. W. Gates, C. F. Petersen, and A. L. Florence; "Laboratory Method for Studying Stemming of Line-of-Sight Tunnels in Underground Nuclear Tests"; Report PYU-2330 (DNA-3592F), December 1973; Stanford Research Institute, Menlo Park, Calif.

5. S. W. Butters and others; "Material Properties of Grouts and of Tuffs from Selected Drill Holes"; Technical Report TR-73-69 (DNA-3383F), July 1974; Terra Tek, Salt Lake City, Utah.

APPENDIX A: NOTATION

C_i	An integration constant in the radial stress solution
G_i	Shear modulus
i	A subscript or superscript referring to a component material in the cross section
K	Effective elastic stiffness
m	Number of materials in the cross section
M	Effective mass
n_i	$N_i - 1$
N_i	$(1 + \sin \phi_i)/1 - \sin \phi_i)$
p(t)	Exterior pressure-time history (compressive)
$p_o(t)$	Interior pressure (compressive)
$P_o(t)$	Effective interior pressure in equation of motion
p_1, p_2	Inside and outside applied load, respectively
q_i	Cohesion
r	Radial coordinate
r_i	Interior boundary of the i^{th} material layer
r_1, r_2	Inside and outside radii, respectively
R	Internal resistance
S^i	Mean normal stress
t	Time
u_c/u_i	Ratio of the compressible to the incompressible radial displacements
u_r^i, u_θ^i	Radial and tangential displacement components, respectively
U	Displacement potential
Y	Effective rigid plastic collapse pressure of the m materials
$\epsilon_r^i, \epsilon_\theta^i$	Radial and tangential strain components, respectively
ν	Poisson's ratio
ρ_i	Density
$\sigma_r^i, \sigma_\theta^i, \sigma_z^i$	Radial, tangential, and axial stress components, respectively
ϕ_i	Angle of internal friction

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A method for designing deep underground structures subjected to dynamic loads, by James L. Drake and James R. Britt. Vicksburg, U. S. Army Engineer Waterways Experiment Station, 1976.
45 p. illus. 27 cm. (U. S. Waterways Experiment Station. Technical report N-76-9)
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References: p. 27.

1. Dynamic loads. 2. Underground structures. I. Britt, James R., joint author. II. Defense Nuclear Agency. III. U. S. Army. Corps of Engineers. (Series: U. S. Waterways Experiment Station, Vicksburg, Miss. Technical report N-76-9)
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